

Disturbance mass: 200g Platform mass: 558 g Platform length: 12 in



Platform and disturbance mass

Plant:

Transfer Function from voltage u to angle θ

Characterized by motor current-to-torque, armature voltage, and load torque relationships, as well as $\theta = \dot{\omega}$

Armature inductance and dynamic friction approximated as zero Parameters found from motor performance curves

$$\frac{\theta}{u} = \frac{\frac{K_t}{R_a}}{Is^2 + K_e \frac{K_t}{R_a}s}$$

 K_t is the motor torque constant

 K_{ρ} is the motor back-EMF constant

 R_a is the armature resistance

I is the platform (unloaded) moment of inertia

Sensor considerations:

Converting angle reading from upside down sensor to true angle UART interference due to computer serial port Allowing time for serial data to clean out before initializing Low sensor noise, but implementation of LPF anyway





Controller Design Methods:

Design CT controller for BW and PM; discretize with ZOH Discretize plant with ZOH, convert to CT with Tustin, design CT controller, then discretize with ZOH.

Design CT controller for BW and PM with ZOH lag accounted for in PM-based calculations

Design CT controller for pole placement & step response, discretize with ZOH

Design for Bandwidth and Phase Margin: PI(D):

Choose a bandwidth and create a family of controllers for a range of phase margins

Some require two poles to generate sufficient phase. These are PID. Two-pole controllers have more aggressive initial response, which is ideal for preventing an object from falling





Design, Modeling, And Control of a Single-Axis Self-Balancing Platform

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Block diagram for control system

Family of step responses for continuous PI(D) controllers with bandwidth \approx 5 rad/s

Comparison of disturbance response for different BW/PM methods (BW = 5; PM = 90°)

Controller:

Discrete Controller

Low-Pass Filter ($\omega_o = 100$) Output Quantized to ±10000 steps due to PWM Output Limited to ±24V

No prefilter needed because of unchanging setpoint (controller designed for disturbance rejection)

Pole Cancellation Considerations:

Little danger in overestimating pole

Underestimating pole could lead to deformations in root locus These deformations are less significant for smaller polar angle and gain



Root locus plot describing effects of cancelling an incorrectly modeled pole

Design for Pole Placement:

One zero at modeled plant fast pole

Choose a polar angle and create a family of controllers for a range of sigmas θ corresponds with overshoot, σ is inversely proportional to settling time If implementing, discretize with ZOH

Overshoots differ because of the inclusion of the LPF



Family of step responses for continuous PI(D) controllers with $\theta \approx 20^{\circ}$



Response with controller θ = 20°, σ = 7



For the purpose of catching a falling mass, a more aggressive transient response is desirable. This can't be increased infinitely because of the limit to voltage supplied.

Response with controller BW = 5, $PM = 90^{\circ}$

Controllers designed for BW and PM did not response similarly to their simulations, but had high enough stability margins to still function

Nonlinearities:

While there was little dynamic friction, static friction is significant With the mass on the end, it would take 5-6 volts for movement to begin Makes it difficult for the system to settle into a steady state gently Torque due to addition of mass

Controllers were able to stabilize for this, but not able to design for this

Next Steps:

- Apply knowledge of software handling of this particular sensor to the controlled ascent of a rocket
- Apply knowledge of software handling of discrete control system (state recording, interrupt timing)
- Use a different motor or find ways to account for static friction in rocket controller design
 - Actuating fins will likely be a lower torque system, especially due to not having to work against gravity, but aerodynamic forces will still cause torques

System will need to be much faster, and overshoot may be less desired

